Abstract—This paper presents the approach to design the Auto-Tuning PID controller for interactive Water Level Process using integral step response. The Integral Step Response (ISR) is the method to model a dynamic process which can be done easily, conveniently and very efficiently. Therefore this method is advantage for design the auto tune PID controller. Our scheme uses the root locus technique to design PID controller. In this paper MATLAB is used for modeling and testing of the control system. The experimental results of the interacting water level process can be satisfyingly illustrated the transient response and the steady state response.

Keywords—Coupled-Tank, Interacting water level process, PID Controller, Auto-tuning.

I. INTRODUCTION

IMORTANTLY, to model the industrial process is necessary to design the linear controller such as PI, PID. There are many methods to model such process for example J.G. Ziegler and N.B. Nichols’s approach [1] as well as K.J Astrom and T. Hugglund’s approach [2] which are famous and better than other techniques. Because of these methods are easy and satisfying to model systems by obtaining the frequency and gain at the critical point of the process. These frequency and gain can be employed to model the process. However, this modeling[2,3] has a divers error with the real process, so that it is bring about designing the better method named Integral System Response (ISR) [4]. The ISR method using the step input signal employs to the process and measures the response from the process for achieving the process parameters.

This paper presents the design of the Auto-Tuning PID controller for interactive Water Level Process by root locus technique and the system modeling can be obtained by integral step response method.

II. THE INTERACTIVE COUPLED-TANK PROCESS

According to Fig. 1, the input $u_1$ is the input pressure which is taken to the pump, and the output $h_2$ is the water level in tank 1. The nonlinear equation can be obtained by mass equilibrium equation and Bernoulli’s law is given by:

$$\frac{dh_1(t)}{dt} = -\frac{\beta_1 a_{12}}{A_1} \sqrt{2g(h_1(t) - h_2(t))} + \frac{k}{A_1} u(t)$$

$$\frac{dh_2(t)}{dt} = -\frac{\beta_2 a_{21}}{A_2} \sqrt{2g h_2(t)} + \frac{\beta_2 a_{12}}{A_2} \sqrt{2g(h_1(t) - h_2(t))}$$

(1)

Fig. 1 The interactive coupled-tank process

Where $A_i$ is the cross section area of tank $i$ ($cm^2$), $a_{ij}$ is the cross section area of outlet of tank 2 ($cm^2$), $\alpha_{ij}$ is the cross section area of jointed pipe between tank 1 and tank 2 ($cm^2$), $\beta_2$ the value ratio at the outlet of tank 2, $\beta_1$ is the value ratio between tank 1 and tank 2, $g$ is the gravity ($cm/s^2$) and $k$ is the gain of pump ($cm^3/Vs$) according to equation 1 is linearized as equation 2.

$$\frac{dH_i(t)}{dt} = \frac{1}{T_i} (-H_i(t) + H_i(t)) + \frac{k}{A_i} u(t)$$

$$\frac{dH_2(t)}{dt} = -\frac{1}{T_2} + H_2(t) + \frac{1}{\alpha_{ij}} (H_1(t) - H_2(t))$$

(2)
Where

\[ T_{12} = \frac{A_1}{\beta_1a_1} \sqrt{\frac{2(h_1 - h_2)}{g}} \], \quad T_s = \frac{A_2}{\beta_2a_2} \sqrt{\frac{2h_2}{g}}.

\( h_1 \) and \( h_2 \) is the water level at operating point of this process, \( T_{12} \) is the time constant between tank 1 and tank 2, and \( T_s \) is the time constant of tank 2 and \( K \). For the equation 2 can be modeled as the equation 3. This is the transfer function for designing this controller.

\[ H(s) = \frac{K}{T_{12}T_s^2 + (T_{12} + 2T_s)s + 1} \]  

(3)

Where

\[ K = kT_s cm/V \]

III. CONTROL SYSTEM STRUCTURE

The control system structure consists of 3 parts as shown in Fig. 2. The first part is the Interactive coupled-tank process; the second part is to define the mathematic model of the process and the two degree of freedom controller.

IV. INTEGRAL SYSTEM RESPONSE

The Integral System Response (ISR) [4] is an effective approach to model the industrial process because this method is able to model such process easily, and the achieved model is very close to the actual process. Formulate the transfer function of the process follows as.

\[ G(s) = \frac{K_c}{\prod_{i=1}^{n} (1 + \tau_i s)} \prod_{i=1}^{n} (1 + \gamma_i s) \]  

(6)

Where

\[ b_i = K_c (\gamma_1 s + \gamma_2 s + \cdots + \gamma_{n-1} s) \quad a_i = (\tau_1 s + \tau_2 s + \cdots + \tau_n s) \]

Input the step signal to the process.

\[ y_i(t) = \int g(t-\tau)(\tau)d\tau \]  

(7)

According to Fig. 3, the steady state response can be achieved and the finite state element can be obtained the equation (7) as following.

\[ \lim_{t \to \infty} y_i(t) = \lim_{s \to 0} \frac{G(s)}{s} = \lim_{s \to 0} \frac{K_c + b_1 s + \cdots + b_{n-1} s^{n-1}}{1 + a_1 s + \cdots + a_n s^n} \]  

(8)

Define by

\[ y_i(t) = \int_0^t (K_0 - y_0(\tau))d\tau \]  

(9)

Where \( y_i(t) \) is the integral area between \( K_0 \) and \( y_0(t) \) to take Laplace to equation (9) as.

\[ Y(s) = \frac{1}{s} [K_0 - G(s)] \]  

(10)

Define by

\[ G_i(s) = \frac{K_0 - G(s)}{s} \]  

(11)
The finite state element can be obtained the equation (10) as following.

$$\lim_{s \to 0} y_1(t) = \lim_{s \to 0} \left[ \frac{K_0 - G(s)}{s^2} \right] = \frac{1}{s} [K_0 - G(s)] = K_0 a_1 - b_1 = K_i$$  \hspace{1cm} (12)

Define by

$$y_1(t) = \int_0^t [K_1 - y_1(t)] d\tau$$  \hspace{1cm} (13)

Where $y_1(t)$ is the integral area between $K_1$ and $y_1(t)$

Fig. 4 The process response when inputs step signal in order to define value for $K_i$

From these equations are able to describe that the high order system must take step signal for many times. The number of taking step input is up to the order of each process which is concluded as the equation below.

$$K_i = K_{ia} a_i - K_{ia} a_2 + K_{ia} a_3 - \cdots + (-1)^{i+1} K_{ia} a_i + (-1)^{i+1} b_j$$  \hspace{1cm} (14)

V. THE ROOT LOCUS TECHNIQUE

To design the controller must be defined the characteristic of transient response and steady state response that can be explained as. [5]

1) The characteristic of transient response can be described in form of percent overshoot (P.O.)

2) The characteristic of steady state response can be described in form of settling time $t_s$

The method to design for satisfying response at the transient state and steady state can be applied as following steps.

Step 1. Finding the damping ratio $\zeta$ and under damped natural frequency: $\omega_n$ by considering the characteristic of transient response and steady state response from the equation (15).

$$P.O. = 100 \% e^{s \pi \sqrt{1-\zeta^2}} \omega_n \quad ts^{(2\pi\alpha)} = 4 / \zeta \omega_n$$  \hspace{1cm} (15)

Step 2. Finding the summation of angle at $s_j$ of the open loop system $G_i(s)G_p(s)$ by graphical method or arithmetical method and then consider the essential angle of $\zeta(s_j + z_j)$ in order to the summation of angle will be being according to the system condition.(16)

$$\sum (\theta_i + \theta_i) + \sum \theta_p = -(2k + 1)\pi, \quad k = 0, 1, \ldots, n$$  \hspace{1cm} (16)

Step 3. Finding the gain $K_e$ of the controller by using the root locus technique.

$$K_e = K_{a0} = \frac{1}{|G(s)|H(s)}$$  \hspace{1cm} (17)

Step 4. Substitution all of the parameters in the equation of controller.

Step 5. Plot the root locus of $G_i(s)G_p(s)$ in order to confirm that the root locus passes the defined point $s_x$.

Step 6. To obtain the satisfying response by inputting step signal therefore, adding the feed forward controller as shown in equation (18).

$$G_f(s) = \frac{z_e}{(s + z_e)}$$  \hspace{1cm} (18)

VI. EXPERIMENT RESULTS

In this paper MATLAB is used for modeling and testing of the control system. The design of the Auto-Tuning PID controller for interactive Water Level Process by root locus technique and the system modeling can be obtained by integral step response method. The experimental results of the interacting water level process can be illustrated the response of the process as shown in Table I and Table II.

<table>
<thead>
<tr>
<th>Table I</th>
<th>THE PARAMETERS OF THE PROCESS</th>
</tr>
</thead>
<tbody>
<tr>
<td>$A_1, A_2; cm^2$</td>
<td>66.25</td>
</tr>
<tr>
<td>$a_1, a_2; cm^2$</td>
<td>0.1963</td>
</tr>
<tr>
<td>$\beta_1$</td>
<td>0.3</td>
</tr>
<tr>
<td>$\beta_1$</td>
<td>0.56</td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>Table II</th>
<th>THE OPERATING POINT OF THE PROCESS</th>
</tr>
</thead>
<tbody>
<tr>
<td>$\bar{h}_1; cm$</td>
<td>9</td>
</tr>
<tr>
<td>$\bar{h}_2; cm$</td>
<td>7</td>
</tr>
<tr>
<td>$u; V$</td>
<td>3</td>
</tr>
<tr>
<td>$k; cm^3/V \cdot s$</td>
<td>2.3</td>
</tr>
</tbody>
</table>

According to the parameters and the operating points of this process can be instead to the equation (3). It will be obtained the transfer function as in equation (19).

$$G(s) = \frac{4.662}{5177s^2 + 307.2s + 1}$$  \hspace{1cm} (19)

VII. THE MODELED PROCESS BY USING ISR METHOD

In the part of modeling system, after comparing the achieved model with the non linear process that can be modeled the process following the ISR method. According to equation (3), the Interactive coupled-tank process, is able to illustrate the second order system with two poles and without zero thus, the modeling method by ISR will be inputted the step signal for three times in order to get value of $K_p, K_i$ and $K_d$ as shown in figure 5, 6 and 7 respectively.
Fig. 5 The process response when inputs step signal for finding $K_0$

Fig. 6 The process response when inputs step signal for finding $K_1$

Fig. 7 The process response when inputs step signal for finding $K_2$

As a result, it will be obtained $K_0$, $K_1$, and $K_2$ as respectively. $K_0 = 5.4373$, $K_1 = 2080.4$ and $K_2 = 772920$. And then the transfer function can be formulated as following.

$$G(s) = \frac{5.4373}{4243.863s^2 + 382.614s + 1}$$

(20)

According to Fig. 8, it illustrates the comparison of the step input response between the modeled system from ISR and nonlinear model. In this article, the transient response of the modeled system from ISR is very similar to the modeled system of nonlinear model, but there is some error not more than 5 percent for ISR method.

VIII. THE STEP INPUT RESPONSE

In this topic, The PID controller design by using root locus technique will be explained. The process model which achieved by ISR as the equation (20) is employed to design the controller under this condition.

$$P.O. \leq 5\%, \omega_s^{(2nd)} \leq 300\text{sec}, e_c(t) = 0$$

From the conditional requirement, it is to be.

$$\zeta = 0.6901, \omega_n = 0.0193, s_d = -0.0133 \pm 0.14$$

And then it is obtained.

$$\theta_1 = 81.0484, z_{c1} = 0.0155, z_{c2} = 0.0885, K_s = 18.2115$$

Therefore, the feedback controller and feed forward controller are able to be shown as following.

$$G_f = \frac{18.2115s^2 + 1.8948s + 0.025}{s}, \quad G_f = \frac{0.0155}{s + 0.0155}$$

As a result, the transient response of the modeled system from ISR has the percent overshoot not more than 5 percent and the setting time is not more than 300 ms. that is under the condition of control system design.
IX. CONCLUSION

This paper presents the design of Auto-tuning PID controller by using Integral System Response method in order to model the process. The interactive Water Level Process used as a case study and the MATLAB is to be a tool for modeling and testing the system. In this article, the transient response of the modeled system from ISR is very similar to the modeled system of nonlinear model, but there is some error not more than 5 percent that is under the condition of control system design.

REFERENCES